

## Summary

The fact that the wear process in a robot joint relates to the increase of friction in the joint is explored:

- wear effects are analyzed and modeled,
- a wear estimator is proposed,
- achievable estimator performance is discussed.

Simulations and experiments show that robust wear estimates are possible even under large temperature variations.

## Friction – Observations and Models

Consider the manipulator rigid-body model:

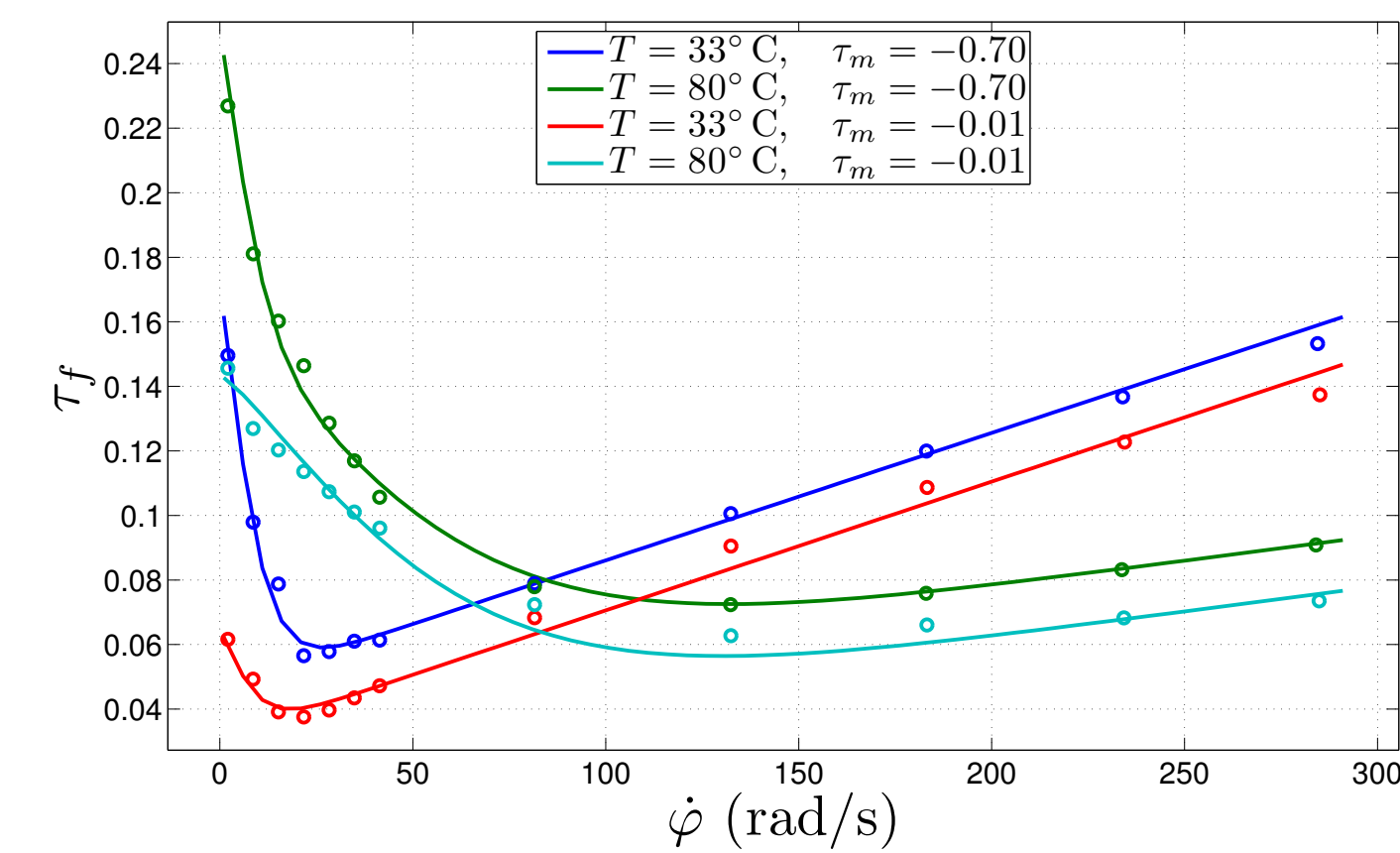
$$M(\varphi) \ddot{\varphi} + C(\varphi, \dot{\varphi}) + \tau_g(\varphi) + \tau_f(\dot{\varphi}) = u$$

Moving one axis at a time in steady-state velocity, then  $\ddot{\varphi} \approx 0$  and  $C(\varphi, \dot{\varphi}) = 0$ .

Take movements over the same position  $\bar{\varphi}$  in forward  $u^+$  and backward  $u^-$  directions, for a constant speed  $\bar{\dot{\varphi}}$ . Under the assumption that  $\tau_f(-\dot{\varphi}) = -\tau_f(\dot{\varphi})$  (direction independence).

$$\begin{aligned} \tau_f(\bar{\dot{\varphi}}) + \tau_g(\bar{\varphi}) &= u^+, \\ \tau_f(-\bar{\dot{\varphi}}) + \tau_g(\bar{\varphi}) &= u^- \end{aligned} \Rightarrow \tau_f(\bar{\dot{\varphi}}) = \frac{u^+ - u^-}{2} \quad (1)$$

Moving the joint through several steady-state velocities back and forth, it is possible to estimate a static friction curve using (1).



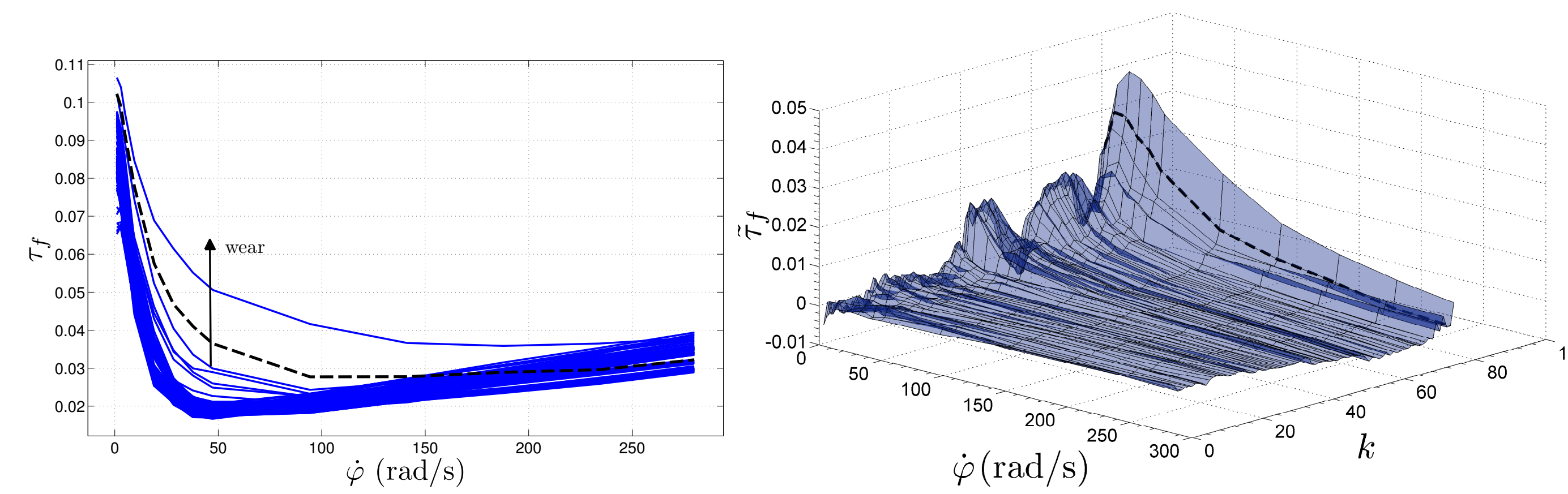
An existing model [1] can predict the static friction with respect to speed, temperature and load.

$$\tau_f(\dot{\varphi}, \tau_m, T) = \{F_{c,0} + F_{c,\tau_m}|\tau_m|\} + F_{s,\tau_m}|\tau_m|e^{-\left|\frac{\dot{\varphi}}{\varphi_{s,\tau_m}}\right|^{1.3}} + \{F_{s,0} + F_{s,T}T\}e^{-\left|\frac{\dot{\varphi}}{\{\varphi_{s,0} + \varphi_{s,T}T\}}\right|^{1.3}} + \{F_{v,0} + F_{v,T}e^{\frac{-T}{T_0}}\}\dot{\varphi}$$

[1] Bittencourt et al. An Extended Friction Model to Capture Load and Temperature Effects in Robot Joints. In IROS 2010, Taiwan, Taipei.

## Wear – Analysis and Modeling

Accelerated wear tests were taken at one of the joints of a 6 axes industrial robot in order to analyze the effects of wear in friction.

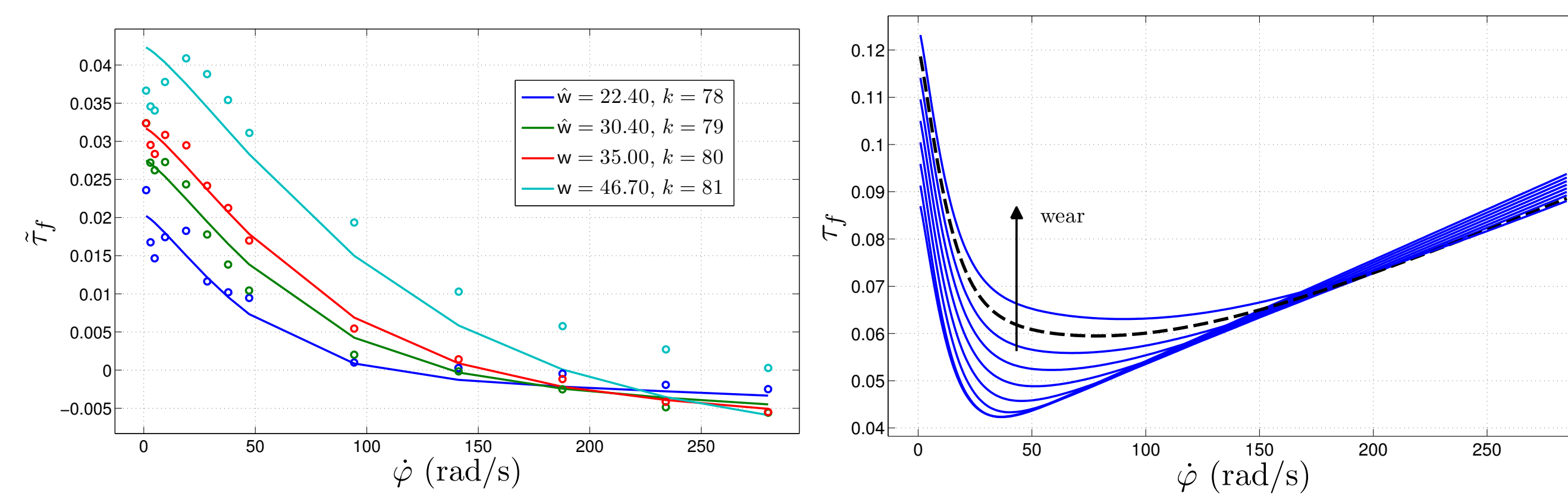


A wear profile  $\tilde{\tau}_f$  is defined as the difference between friction curves under no wear and the remaining ones.

Introducing  $\mathbf{w}$ , with values between  $[0, 100]$ , a model is proposed for  $\tilde{\tau}_f$ .

$$\tilde{\tau}_f(\dot{\varphi}, \mathbf{w}) = F_{s,\mathbf{w}}\mathbf{w}e^{-\left|\frac{\dot{\varphi}}{\varphi_{s,\mathbf{w}}}\right|^{1.3}} + F_{v,\mathbf{w}}\mathbf{w}\dot{\varphi}$$

The parameters for the wear profile are identified with the convention that  $\mathbf{w} = 35$  at  $k = 80$ .



The existing model can be extended by simply adding the wear profile  $\tilde{\tau}_f$ .

$$\tau_f(\dot{\varphi}, \tau_m, T, \mathbf{w}) = \tau_f(\dot{\varphi}, \tau_m, T) + \tilde{\tau}_f(\dot{\varphi}, \mathbf{w})$$

The resulting model can be used for model-based wear identification under broad operation conditions.

$$\hat{\mathbf{w}} = \arg \min_{\mathbf{w}} V(\tau_f - \hat{\tau}_f(\dot{\varphi}, \tau_m, T, \mathbf{w})),$$

Joint temperature measurements are however not possible in usual industrial applications, but should be distinguished from wear effects.

## A model-based Wear Estimator

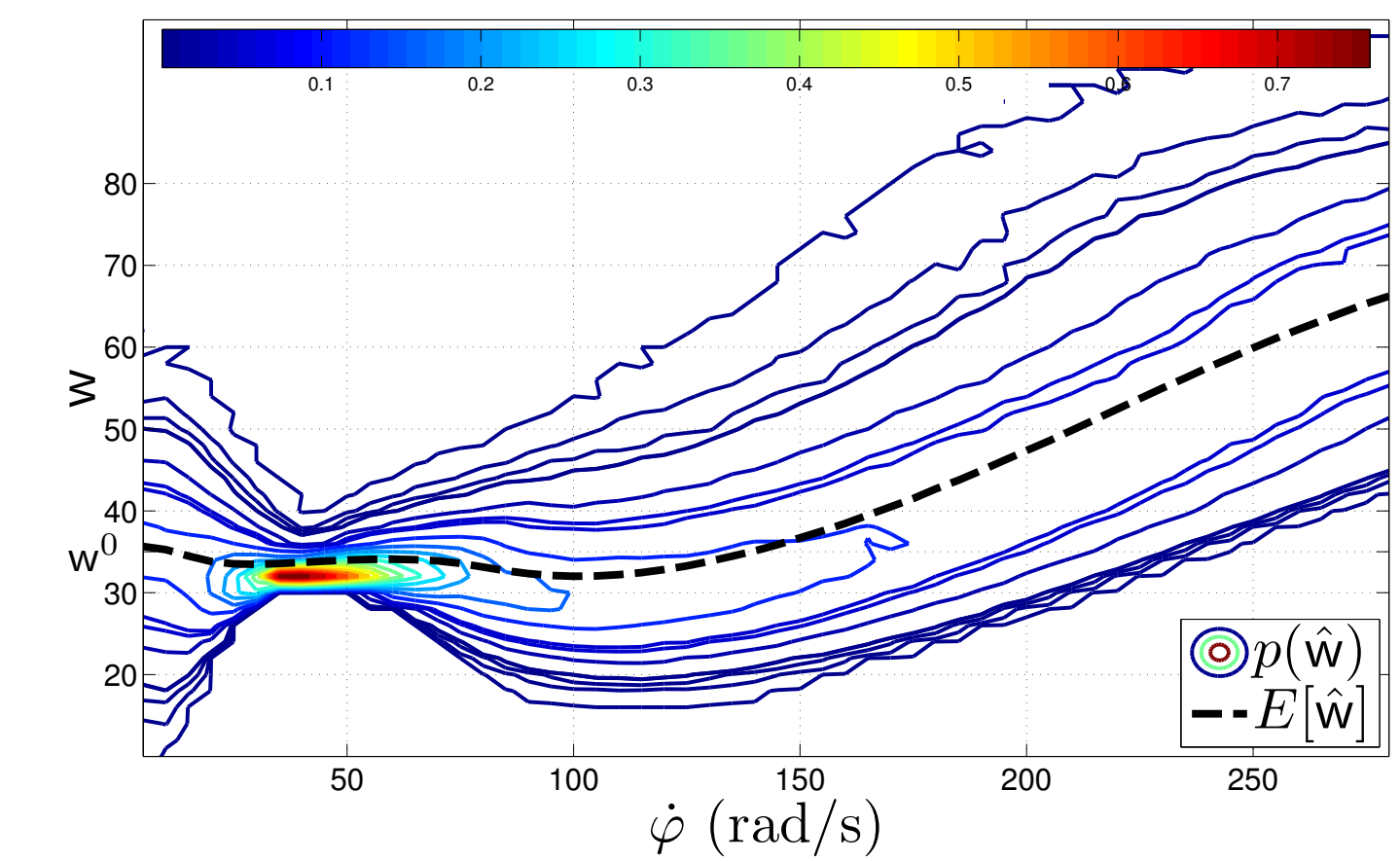
The estimator assumes  $T$  uniformly distributed between the limits and  $N$  samples are drawn from it. The expected value of the resulting  $N$  estimates  $\{\hat{\mathbf{w}}_N\}$  is then taken as the wear estimate.

$$\hat{\mathbf{w}}_i = \arg \min_{\mathbf{w}} V(\tau_f - \hat{\tau}_f(\dot{\varphi}, \tau_m, T_i, \mathbf{w})) \quad (2a)$$

$$\hat{\mathbf{w}} = E[\{\hat{\mathbf{w}}_N\}], \quad T_i \sim \mathcal{U}(\underline{T}, \overline{T}), \quad i = 1, \dots, N \quad (2b)$$

**Simulation.** A Monte Carlo simulation study is performed for the estimator when the observed friction  $\tau_f$  has  $\mathbf{w} = 35$  and  $T \sim \mathcal{N}(40, 3)$  and the estimator is set with  $N = 200$ ,  $\underline{T} = 30^\circ\text{C}$  and  $\overline{T} = 50^\circ\text{C}$ .

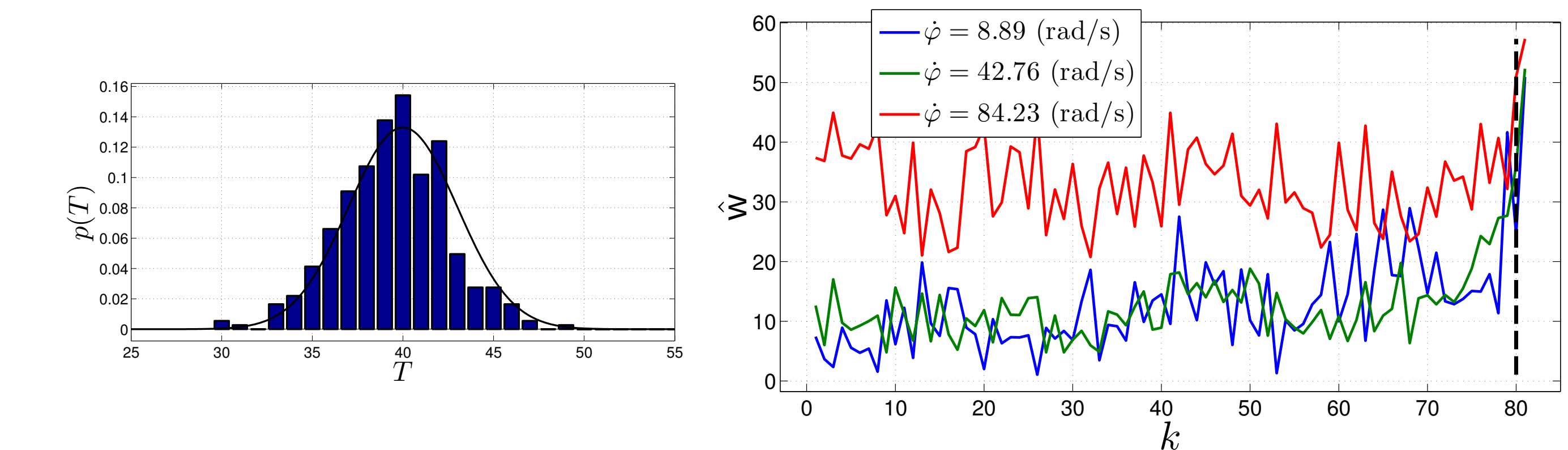
- Large bias at high  $\dot{\varphi}$ .
- Large var. at low/high  $\dot{\varphi}$ .
- Selective  $\dot{\varphi}$  region where  $\hat{\mathbf{w}}$  estimates are useful.



**Case Study.** Data from a wear profile  $\tilde{\tau}_f$  is added to friction curves from a normal robot  $\tau_f^0(T)$  under several  $T$  conditions.

$$\tau_f^*(k) = \tilde{\tau}_f(k) + \tau_f^0(T) \quad (3)$$

The data for  $\tau_f^0(T)$  are sampled according to a desired temperature distribution and  $\mathbf{w}$  is estimated.



With the estimates of  $\mathbf{w}$  at  $\dot{\varphi} = 42.76$  rad/s, even a simple threshold set at  $\mathbf{w} = 35$  could detect the friction increase at a critical wear level (indicated by the dashed line).

**For more.** This work is available as a technical report at <http://www.control.isy.liu.se/~andrech/publications/>